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# Use of Principle of Contra-Gradience for Developing Flexibility Matrix 

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#### Abstract

Flexibility method and stiffness method are the two basic matrix methods of structural analysis. Linear simultaneous equations written in matrix form are always easy to solve. Also when computers are used, matrix algebra is very convenient.

When static degree of indeterminacy is less than the kinematic degree of indeterminacy, flexibility method is advantageous in structural analysis. The final step in flexibility method is to develop total structural flexibility matrix. The new approach to develop total structural flexibility matrix, requires forcedeformation relation, transformation matrix and principal of Contra-gradience. Subsequently compatibility equations are required to get the value of unknown redundants. This new approach while using flexibility method is useful because simple matrix multiplications are required as well as order of the matrices does not become very large.

The application of this new approach to flexibility method is illustrated by solving few problems of structural analysis. MATLAB software is used to do matrix calculations. The results obtained by this new approach are compared with STAAD Pro results.


Key Words: Principle of Contra-gradience, MATLAB, STAAD Pro.v8i, Flexibility matrix

## 1.INTRODUCTION

In the flexibility method, the primary structure is the released form, where unknown actions have been released. The primary unknowns are the released unknown actions. A significant feature of the method is that the analyst has a choice in the selection of the action to be released. Flexibility coefficients giving the deformation values due to unit action applied in the direction of primary unknowns are used in the new approach of flexibility method along with force and deformation transformation technique. And those flexibility coefficients can be developed by using different methods of structural analysis for example unit load method, theorem of three moments, slope deflection equations.

This new method is used when static degree of indeterminacy is less than kinematic degree of indeterminacy. And order of matrix required in this new method is equal to static degree of indeterminacy. So order of matrices does not become very large which will occupy less memory of computer if program is made by using this new approach of flexibility method. The need of developing
computer program based of flexibility method is increased because the procedure of the direct stiffness method is so mechanical that it risks being used without much understanding of the structural behaviours.

Matrix algebra required in new approach of flexibility method is also very simple. So the time required for manual calculations can be reduced by using MATLAB software.

### 1.1 Principle of Contra-gradience

Consider the two co-ordinate systems 0 and 0 ' as shown in fig 1 . The forces acting at co-ordinate system 0 are $F_{X}, F_{Y}$ and M and the forces acting at co-ordinate system $\mathrm{O}^{\prime}$ are $F_{X}^{\prime}, F_{Y}^{\prime}$ and $M^{\prime}$. Here we are interested to transfer the forces from co-ordinate system 0 to $0^{\prime}$. And the effect of this force transformation on co-ordinate system $0^{\prime}$ can be given as


Figure 1. Force Transformation

$$
\begin{align*}
& {\left[\begin{array}{c}
F_{X}^{\prime} \\
F_{Y}^{\prime} \\
M^{\prime}
\end{array}\right]=\left[\begin{array}{ccc}
1 & 0 & 0 \\
0 & 1 & 0 \\
k & -h & 1
\end{array}\right]\left[\begin{array}{c}
F_{X} \\
F_{Y} \\
M
\end{array}\right]}  \tag{1}\\
& {\left[F^{\prime}\right]=\left[T_{f}\right][F]}
\end{align*}
$$

Where $\left[T_{f}\right]$ is the force transformation matrix from coordinate system 0 to $0^{\prime}$.Now again consider the two coordinate systems 0 and $0^{\prime}$. The deformations occurring at coordinate system 0 are $\partial_{X}, \partial_{Y}$ and $\theta$. And the deformations occurring at co-ordinate system $0^{\prime}$ are $\partial_{X}^{\prime}, \partial_{Y}^{\prime}$ and $\theta^{\prime}$. Now we are interested to transfer the deformation from coordinate system 0 ' to 0 . And the effect of this deformation transformation on co-ordinate system 0 can be given.


Figure 2. Deformation Transformation

$$
\begin{align*}
& {\left[\begin{array}{c}
\partial_{X} \\
\partial_{Y} \\
\theta
\end{array}\right]=\left[\begin{array}{ccc}
1 & 0 & k \\
0 & 1 & -h \\
0 & 0 & 1
\end{array}\right]\left[\begin{array}{l}
\partial_{X}^{\prime} \\
\partial_{Y}^{\prime} \\
\theta^{\prime}
\end{array}\right]}  \tag{2}\\
& {[\partial]=\left[T_{\partial}\right]\left[\partial^{\prime}\right]}
\end{align*}
$$

Where $\left[T_{\partial}\right]$ is the deformation transformation matrix from co-ordinate system 0 ' to 0 .
It is seen from equation (1) and (2) above

$$
\left[T_{f}^{T}\right]=\left[T_{\partial}\right]
$$

Transpose of force transformation matrix from co-ordinate system A to B gives displacement transformation matrix from co-ordinate system B to A. This is known as Principle of Contra- gradience.

### 1.2 Nodal Deformation Method

Figure 3 shows rigid jointed frame PQR with values of flexural rigidity (EI), axial force rigidity (EA) and length (L) for both members PQ and QR.


Figure 3. Frame PQR
Total deformation occurring at R is caused by mainly the deformations of both members PQ and QR. And for that flexibility coefficients are used which will give deformations due to the unit action. Total deformation can be given as

$$
[F]=\left[F_{1}\right]+\left[F_{2}\right]
$$

where $\quad[F]=$ Flexibility matrix for the frame $P Q R$ $\left[F_{1}\right]=$ Deformation due to member PQ
$\left[F_{2}\right]=$ Deformation due to member QR


Figure 4. free body diagram
From the equilibrium of member QR ,
$\left\{\begin{array}{l}Y_{1} \\ Y_{2} \\ Y_{3}\end{array}\right\}=\left[\begin{array}{ccc}0 & 1 & 0 \\ 1 & 0 & 0 \\ L_{2} & 0 & 1\end{array}\right]\left\{\begin{array}{l}X_{1} \\ X_{2} \\ X_{3}\end{array}\right\}$
OR $\{Y\}=\left[T_{Q R}\right]\{X\}$
Where $\{Y\}$ and $\{X\}$ are the column matrices of the actions Q and R respectively. $\left[T_{Q R}\right]$ is the transformation matrix for the actions at $Q$ from R. Hence

$$
\left[F_{1}\right]=\left[T_{Q R}^{T}\right]\left[F_{P Q}\right]\left[T_{Q R}\right]
$$

Where $\left[F_{P Q}\right]$ is the flexibility matrix for unit actions applied in the direction of $\{Y\}$ for member $P Q$ and is given by

$$
\left[F_{P Q}\right]=\left[\begin{array}{ccc}
\frac{L_{1}^{3}}{3 E I_{1}} & 0 & \frac{L_{1}^{2}}{2 E I_{1}} \\
0 & \frac{L_{1}}{E A_{1}} & 0 \\
\frac{L_{1}^{2}}{2 E I_{1}} & 0 & \frac{L_{1}}{E I_{1}}
\end{array}\right]
$$

Similarly $\left[F_{2}\right]=\left[F_{Q R}\right]=\left[\begin{array}{ccc}\frac{L_{2}^{3}}{3 E I_{2}} & 0 & \frac{L_{2}^{2}}{2 E I_{2}} \\ 0 & \frac{L_{2}}{E A_{2}} & 0 \\ \frac{L_{2}^{2}}{2 E I_{2}} & 0 & \frac{L_{2}}{E I_{2}}\end{array}\right]$
Thus the flexibility matrix for the whole frame can be developed by using flexibility coefficients along with force and deformation transformation technique as well as principle of contra-gradience. Following examples will show the use of this new approach of flexibility method.

## 2. PROBLEMS AND ANALYSIS

Example (1): The fixed beam shown in figure 5 where the length of each member is L and it is subjected to a concentrated force P. There is step variation of the flexural rigidity EI.


Figure 5. Fixed Beam
Static degree of indeterminacy $=2$
The reaction R and moment M at fixed end D are considered as redundant as shown in figure 6.


Figure 6. Free body diagram of fixed beam

 where,

$$
\begin{aligned}
& {\left[T_{C D}\right]=\left[\begin{array}{ll}
1 & 0 \\
L & 1
\end{array}\right]} \\
& {\left[\mathrm{F}_{\mathrm{CD}}\right]=\left[\begin{array}{cc}
\frac{\mathrm{L}^{3}}{6 \mathrm{EI}} & \frac{\mathrm{~L}^{2}}{4 \mathrm{EI}} \\
\frac{\mathrm{~L}^{2}}{4 \mathrm{EI}} & \frac{\mathrm{~L}}{2 \mathrm{EI}}
\end{array}\right]=\left[\mathrm{F}_{\mathrm{AB}}\right]}
\end{aligned}
$$

$$
\left[\mathrm{F}_{\mathrm{BC}}\right]=\left[\begin{array}{cc}
\frac{\mathrm{L}^{3}}{3 \mathrm{EI}} & \frac{\mathrm{~L}^{2}}{2 \mathrm{EI}} \\
\frac{\mathrm{~L}^{2}}{2 \mathrm{EI}} & \frac{\mathrm{~L}}{\mathrm{EI}}
\end{array}\right]
$$

By solving above equation, we will get

$$
\begin{gathered}
{\left[\begin{array}{ll}
F_{A D}
\end{array}\right]=\left[\begin{array}{ll}
1 & l \\
0 & 1
\end{array}\right]\left\{\left[\begin{array}{ll}
1 & L \\
0 & 1
\end{array}\right]\left[\begin{array}{cc}
\frac{L^{3}}{6 E I} & \frac{L^{2}}{4 E I} \\
\frac{L^{2}}{4 E I} & \frac{L}{2 E I}
\end{array}\right]\left[\begin{array}{ll}
1 & 0 \\
\frac{0}{2} & 1
\end{array}\right]+\left[\begin{array}{cc}
\frac{L^{3}}{3 E I} & \frac{L^{2}}{2 E I} \\
\frac{L^{2}}{2 E I} & \frac{L}{E I}
\end{array}\right)\right\}\left[\begin{array}{ll}
1 & 0 \\
\frac{L}{L} & 1
\end{array}\right]+\left[\begin{array}{cc}
\frac{L^{3}}{6 E I} & \frac{L^{2}}{4 E I} \\
\frac{L^{2}}{4 E I} & \frac{L}{2 E I}
\end{array}\right]} \\
{\left[\begin{array}{ll}
\frac{17 L^{3}}{3 E I} & \frac{3 L^{2}}{E I} \\
\frac{3 L^{2}}{E I} & \frac{2 L}{E I}
\end{array}\right]}
\end{gathered}
$$

Compatibility condition at the point D is given by,

$$
\left[F_{A D}\right]\binom{R}{M}+\left[\mathrm{T}_{\mathrm{CD}}\right]^{\mathbb{T}}\left\{\left[\mathrm{T}_{\mathrm{BC}}\right]^{\mathrm{T}}\left[F_{A B}\right]\left[T_{B C}\right]+\left[F_{B C}\right]\right\}\binom{P}{0}=\binom{0}{0}
$$

Calculations - solving manually,

$$
\binom{R}{M}=\binom{3 / 4}{-L / 2} P
$$

In Example 1:
Put $\mathrm{L}=3 \mathrm{~m}$ \& point load $\mathrm{P}=10 \mathrm{kN}$
By solving manually, results obtained are as follows

$$
\begin{aligned}
& \mathrm{R}=7.5 \mathrm{kN} \\
& \mathrm{M}=-15 \mathrm{kN} \mathrm{~m}
\end{aligned}
$$

$>$ CALCULATIONS -BY SOFTWARE (STAAD PRO) APPROACH:


Table- 1. STAAD PRO RESULT

|  |  | Horiz <br> ontal | Vertical | Horizont <br> al | Moment kNm |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Node | Load <br> case | Fx kN | Fy kN | Fz kN | Mx | My | Mz |
| 3 | $\mathrm{P}=10$ <br> kN | 0.000 | 7.495 | 0.000 | 0.000 | 0.00 | 14.986 |
| 4 | $\mathrm{P}=10$ <br> kN | 0.000 | 2.505 | 0.000 | 0.000 | 0.00 | 7.535 |

> Calculations -by Using MATLAB SOFTWARE MATLAB INPUT
(i) New to MATLAB? Watch this Video, see Demos, or read Getting Started.
$\gg$ Tf $=[1,0 ; 3,1]$
Tf $=$
10
>> fbc=[9,4.5;4.5,3]
$\mathrm{fbc}=$
$9.0000 \quad 4.5000$
$4.5000 \quad 3.0000$
$\gg f a b=[4.5,2.25 ; 2.25,1.5]$
$\mathrm{fab}=$
$4.5000 \quad 2.2500$
$2.2500 \quad 1.5000$
>> fcd=[4.5,2.25;2.25,1.5]
fcd $=$
$4.5000 \quad 2.2500$
$2.2500 \quad 1.5000$

## MATLAB OUTPUT

```
>> Fbc=TE"*Eab*TE+Ebc
FbC =
        40.5 11.25
        11.25 4.5
>> Fad=TE"*Fbc*TE+ECd
Fad =
    153 27
>>C=TE:*Fbc*[10:0]
c =
742.5
112.5
\(\rightarrow\) ans=Fadn-1*C
ans \(=\)
```

$$
\begin{aligned}
& 7.5 \\
& -15
\end{aligned}
$$

From above results, manual results are exactly matching with STAAD PRO results and MATLAB software can be used effectively for calculations.

Example (2): A rigid jointed frame ABCDEFGHIJ is shown in figure 8. The flexural rigidity EI is uniform for the whole frame. Each element is of the same length L. The frame is subjected to a concentrated force $P$ at the intermediate point E.


Figure 8. Frame ABCDEFGHIJ


Figure 9. Free Body Diagram
The given frame from A to E and E to I are identical. The change in the length of the members is to be neglected. This makes the degree of kinematic indeterminacy equal to 15 and the degree of static redundancy is 3 . The three end actions at J, namely $\mathrm{V}, \mathrm{H}$ and M are considered redundant. By new method using Nodal Deformation and Principle of Contra-gradience, the equation can be written as,

Thus

$$
\begin{aligned}
& +\left[\begin{array}{lll}
\frac{L^{3}}{3} & 0 & \frac{L^{2}}{3} \\
0 & 0 & 0 \\
\frac{L^{2}}{2} & 0 & L
\end{array}\right] ;\left[\begin{array}{ccc}
0 & -1 & 0 \\
1 & 0 & 0 \\
2 & 0 & 1
\end{array}\right]+\left[\begin{array}{lll}
\frac{L^{3}}{\frac{1}{3}} & 0 & \frac{L^{2}}{2} \\
0 & 0 & 0 \\
\frac{L^{2}}{2} & 0 & L
\end{array}\right]
\end{aligned}
$$

By solving manually,
$\left[F_{A E}\right]=\left[\begin{array}{ccc}\frac{23 L^{a}}{3} & -3 L^{a} & 5 L^{2} \\ -3 L^{a} & \frac{5 L^{a}}{3} & -2 L^{2} \\ 5 L^{2} & -2 L^{2} & 4 L\end{array}\right]$
Since $\left[F_{A E}\right]=\left[F_{E J}\right]$
The compatibility condition at the end J is written as $\left.\left\{\left[F_{I I}\right]+\left[T_{T I}\right]^{\mathrm{T}}\left[T_{E I I}\right]^{\mathrm{T}}\left[F_{A B}\right]\left[T_{E l}\right]\left[T_{I I}\right]+\left[T_{\Pi}\right]\right]^{\mathrm{T}}\left[F_{E I I}\right]\left[T_{I I}\right]\right\}\left(\begin{array}{l}\mathrm{V} \\ \mathrm{H} \\ \mathrm{M}\end{array}\right)+\left[\mathrm{T}_{\Pi}\right]^{\mathrm{T}}\left[T_{E I I}\right]^{\mathrm{T}}\left[F_{A B}\right]\left(\begin{array}{l}\mathrm{P} \\ 0 \\ 0\end{array}\right)=\left(\begin{array}{l}0 \\ 0 \\ 0\end{array}\right)$
Yielding the answer
$\left(\begin{array}{l}V \\ H \\ M\end{array}\right)=\frac{P}{242}\left[\begin{array}{c}-145 \\ -18 \\ 166 L\end{array}\right]$

Put L= 3 m \& point load $\mathrm{P}=10 \mathrm{kN}$.
By solving manually, results obtained are as follows

$$
\begin{aligned}
& \mathrm{V}=0 \mathrm{kN}, \\
& \mathrm{H}=5 \mathrm{kN} \\
& \mathrm{M}=13.33 \mathrm{kN} \mathrm{~m}
\end{aligned}
$$

CALCULATIONS: BY SOFTWARE (STAAD Pro) APPROACH


Figure 10. STAAD Pro MODEL OF 9 MEMBER FRAME

Table - 2 STAAD Pro RESULTS

|  |  | Horiz <br> ontal | Vertic <br> al | Horizont <br> al | Moment <br> kNm |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Node | Load <br> case | Fx kN | Fy kN | Fz kN | Mx | My | Mz |
| $\mathbf{1}$ | $\mathrm{P}=$ <br> 10 kN | 0.000 | 5.001 | 0.000 | 0.00 | 0.000 | 13.338 |
| $\mathbf{1 0}$ | $\mathrm{P}=$ <br> 10 kN | 0.00 | 4.999 | 0.000 | 0.00 | 0.000 | -13.329 |

## 3. CONCLUSIONS

The application of new approach of flexibility method is shown here by solving examples. Results obtained from manual calculations are exactly matching with software results. So flexibility coefficients of individual members along with force and deformation transformation technique and principle of Contra-gradience can be used to develop the flexibility matrix for the whole structure. Also MATLAB software can be used to do mathematical calculations.

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