

REVIEW OF TRUE BENDING STRESS IN SPUR GEAR

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Abstract - Gears are utilized to carry power starting from driver shaft then onto the driven when they are separated by narrow distance. They keep up steady speed proportion overcoming slip. Helical, Spur, rack and pinion type, worm and worm wheel, bevel, etc. Gears having teeth lateral to axis are spur gears most accepted, because they made effortlessly also equipped for sustaining typical loading conditions and useful for small speeds. Lots of spur gears utilized at once for maximum speed reduction. Every time when spur gear tooth came in contact with the other one, they collides each other and due to this collision it make lots of sound and also maximize the stress induced.

Key Words: Helical, Spur, rack and pinion, worm and worm wheel, bevel, slip.

1. INTRODUCTION

Gears are important amongst wide basic segments in force transfer mechanism, also in maximum modern rotating apparatus. We can consider that gear will prevail as best method for transmitting power in future machines because of their high level quality of power transmission and smallness. Moreover, the quick move in the business from overwhelming enterprises, for example, shipbuilding to ventures, for example, vehicle production and office robotization instruments will require a refined utilization of gearing innovation.

Gears examinations in the past were carried out applying analytical technique, which required various presumptions and improvements. As a rule, gear examinations are multidisciplinary, including determination of tooth stress and Failure, for example, similar to erode or scuffing. Here, fixed contact and bending stress examinations are carried out, although attempting to model the gear which can sustain the bending stress failure and pitting, as both influence transmission error. As computer technics have turned out to be increasingly intense, individuals have tended to utilize numerical ways to deal with create hypothetical prototype to anticipate impact of anything that contemplated. It enhanced gear analysis also computer simulations. Analytical techniques also conceivably give deeper precise arrangements after all it ordinarily needs a great deal less prohibitive suppositions. The model and the arrangement strategies, in any case, must be picked deliberately to guarantee that the outcomes are exact and that the computational time is sensible.

2. TRUE BENDING STRESS

It can be considered from literary works that 'true bending stress' at root of tooth in a spur gears is not same as that of the supposed or theoretical values that are used for the determining the load capacity of spur gear, either by principals or common designing laws [4]. No issues emerge in utilizing a weight holding strength while standards correlated along the outcomes of fatigue tests on bending for which cutoff points determined along similar simplified technics. In any case, at tooth root true stress has dissimilar pattern plus standards, plus the creator need to know this dissimilar pattern, particularly considering portable gears along small ribs plus rims.

On account of strength of tooth to sustain bending, cantilever-beam prototype most widely utilized to determine the bending intensity. Using this methodology, during 1892 Lewis initially figured spur gear root stress of teeth. This model is a base for standard computation techniques effectively utilized as a part of gear design. In any case, the regional stress condition true load intensity along tooth root fillet might be not same as the theoretical values got using this technique. In real, the determination of the maximum extension stress near to tooth root is 3-D issue: The plane stress or plane strain prototype can be adopted excluding guesstimate just as a part of the instance of infinite, or infinitesimal, face width.

For this project, exact Finite Element Method analyses completed on true stress near root for spur gears as a role of gear geometry. This results got affirm significance for those distinctions.

3. BENDING STREESES

There are a few types of fracture modes in spur gears, and one of the fundamental or important type of facture modes is 'Bending failure' of the teeth [11]. The bending stresses are another interesting issue in a spur gear. When a load is maximum, bending failure will take place. In gears bending failure is anticipated by correlating the numerically determined to practically calculate permissible fatigue standards of assumed material. The equation for bending stress got using Lewis expression.

Distinct elements that are needed for the computation can be taken from the machine design books. This examination considers just the segment of tooth where only tangential force is acting, and it does not consider outward load, that results in compressive stress on roots cross section of tooth. Assume maximum stress results only while load applied on the upper side of tooth, that's most pessimistic scenario. While force on upper side tooth, for the most part there are no less than 2 tooth sets in engage. The greatest stress at the root happens while engage point travels close to pitch circle on the grounds that there is one and only tooth pair in contact and these teeth sets conveys the whole torque. At the point when the force is moving at the highest point of the tooth, two teeth sets share the entire load if the proportion is bigger than one and under two. In the event that only tooth pair assumed to convey entire force plus that follows up highest point for tooth that sufficient to gear bending stress fatigue.

In gear tooth catastrophic failure or yielding is due to extreme bending stress that's why it is one of essential gear design concept. With a specific end goal to anticipate catastrophic failure or yielding, we need to determine both the tensile and compressive sides greatest stress on the tooth separately. First of all, the photo elasticity or relatively coarse FEM meshes is used to determine bending stress sensitivity of a gear tooth. But, now a day's due to software and computer advancements we can make critical changes for more precise FEM simulations.

4. LITERATURE SURVEY

Wellauer and Seireg [5] presented an investigation of the gear teeth's bending stress using a cantilever-plate model and the strategy demonstrates obviously that a 3-D model utilized to assess the variety of root stress on the tooth along the face width. It gives a research on bending strength on teeth of gear using cantilever-plate model. A semi observational arrangement is given for the limited cantilever plate under transverse burdens at any area on its surface. The arrangement which depends on the standard of superposition and a proposed minute picture strategy demonstrated great concurrence with results from straingage examinations on cantilever plates reenacting the apparatus tooth. The impacts of the position of the line load and the variety of burden power on the anxiety dissemination at the root are examined.

Zeping Wei, [1] in a spur gear deflection and stresses at the root is determined by two & three dimensional ANSYS geometry. Here compared results of two dimensional analysis & three dimensional analysis. This proposition explores the attributes of an involute apparatus framework including contact stresses, twisting burdens, and the transmission

blunders of apparatuses in cross section. Equipping is a standout amongst the most basic segments in mechanical force transmission frameworks. Transmission blunder is thought to be one of the principle patrons to clamor and vibration in an apparatus set. Transmission mistake estimation has ended up mainstream as a region of examination on apparatuses and is conceivable technique for quality control. To gauge transmission mistake in a gear framework, the attributes of involute goad riggings were examined by utilizing the limited component strategy. The contact burdens were analyzed utilizing 2-D FEM models. The twisting hassles in the tooth root were inspected utilizing a 3-D FEM model. Current techniques for ascertaining gear contact stresses utilize Hertz's conditions, which were initially determined for contact between two chambers.

Klenz [6] spur gear pair examined utilizing Finite Element Method. The stress at the point of engage is inspected utilizing 2-D Finite Element Method prototype. On various small rimmed gears analyses for bending stress was performed. In his paper he gives study on examination of engage point stress plus bending stress.

Tsay [3] has considered helical gears in which he determined bending plus engage stresses utilizing FEM by tooth engage analysis system.

Mrs. S.P.Shinde, Mr. A.A. Nikam & Mr. T.S. Mulla [7] -Rigging is a standout amongst the most basic part in a mechanical force transmission framework, and most modern pivoting hardware. A couple of goad apparatus teeth in real life is for the most part subjected to two sorts of cyclic hassles: bowing burdens initiating twisting exhaustion and contact stress bringing on contact weakness. Both these sorts of burdens may not achieve their most extreme qualities at the same purpose of contact weariness. These sorts of disappointments can be minimized via watchful investigation of the issue amid the outline arrange and making legitimate tooth surface profile with appropriate assembling strategies. When all is said in done, gear examination is multidisciplinary, including estimations identified with the tooth focuses and to tribological disappointments, for example, wear or scoring. In this paper, bowing anxiety examination will be performed, while attempting to outline goad apparatuses to oppose twisting disappointment of the teeth, as it influences transmission blunder. To begin with, the limited component models and arrangement techniques required for the precise computation of bowing anxieties will be resolved. At that point twisting burdens computed utilizing ANSYS, were contrasted with the outcomes acquired from existing techniques.

Krishanu Gupta & Sushovan Chatterjee [8] - The guideline target of this paper is the examination investigation of the static hassles for goad gear with various weight edges. The investigated consequences of a symmetric sort involute profiled goad gear pair at various weight points are looked at. Apparatuses are a standout amongst the most essential and



significant segment in a mechanical force transmission unit furthermore in the majority of the modern turning apparatuses. For the most part, a goad gear pair in real life experiences two sorts of anxieties: the twisting anxiety and the contact stress. In this paper, both these weights on the gear tooth pair are broke down utilizing the limited component investigation and are looked at. The weights on the rigging tooth are initially examined utilizing a limited component programming and afterward those outcomes are accepted utilizing the customary formulae for discovering stresses in rigging tooth.

5. CONCLUSION

Reviving all the literature Surveys we conclude that the maximum stress in gear is located at the tooth root of gear. The maximum stress is in the middle zone of the thickness whereas on the sides the stresses are lesser. The bending stress is varies linearly with increase in Face width of the gear.

REFERENCES

- [1] Zeping Wei, "Stresses & Deformation in Involute Spur Gears by Finite Element Method", Department of Mechanical Engineering, University of Saskatchewan, Saskatoon, Saskatchewan,-Thesis, 2004.
- [2] Lecture Notes, "Spur Gear Design" IIT Madras, India
- [3] EdoardoConrado and PiermariaDavoli, "The True Bending Stress In Spur Gears", Gear Technology, August 2007
- [4] Anil Patil, "Analysis of Bending Stress at Root of the Spur Gear by Finite Element Method" Shantiniketan Polytechnic, Maharashtra
- [5] Joseph Edward Shigley, Charles R. Mischke"Mechanical Engineering Design", McGraw-Hill Book Company, New York, 2000, Sixth Edition
- [6] Lecture Notes, "Spur Gear Design" IIT Madras, India
- [7] Mrs. S.P.Shinde, Mr. A.A. Nikam & Mr. T.S. Mulla "Static Analysis of Spur Gear Using Finite Element Analysis" Dept. of Mechanical Engg, Dr.J.J.Magdum College Of Engg. Jaysingpur, India. ISSN: 2278-1684, PP: 26-31
- [8] Krishanu Gupta & Sushovan Chatterjee "Effect of pressure angle of spur gears on bending and contact stresses" Department of Mechanical Engineering, NIT Silchar (India), March 2015. ISSN-2319-8354(E)

- [9] Ali Kamil Jebur, I.A.Khan, Y.Nath "Numerical and Experimental Dynamic Contact of Rotating Spur Gear" Department of mechanical engineering jamia millia islamia new delhi-110025, India, Vol. 5, No. 2; April 2011 254 ISSN
- [10] Mr. A. Gopichand, Prof. A.V.N.L. Sharma, K. Pavan Kumar, K. Sainath, I. Aravind "Design of Spur Gear and its Tooth profile" ISSN: 2248-9622 www.ijera.com Vol. 2, Issue 2, Mar-Apr 2012, pp.820-827
- [11] Dhavale A.S., Abhay Utpat "Study of Stress Relief Features at Root of Teeth of Spur Gear" Department of Mechanical, S.V.E.R.I. COE Pandharpur, India Vol. 3, Issue 3, May-Jun 2013, pp.895-899, ISSN: 2248-9622
- [12] Vivek Karaveer, Ashish Mogrekar and T. Preman Reynold Joseph "Modeling and Finite Element Analysis of Spur Gear" SMBS Department, VIT University, Chennai, Tamilnadu, India, 30 December 2013, Vol.3, No.5, ISSN 2277 - 4106

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